

# Lagrangian formulation of mechanics

Newton's laws can be restated in a very elegant form using minimum principles like Fermat's principle of least time.

Furthermore, this approach is very convenient because it introduces the idea of "generalized coordinates"  $q_i$  ( $i=1, \dots, n$ ) and "generalized velocities"  $\dot{q}_i$  ( $i=1, \dots, n$ ) - N.B. these need not be cartesian coords

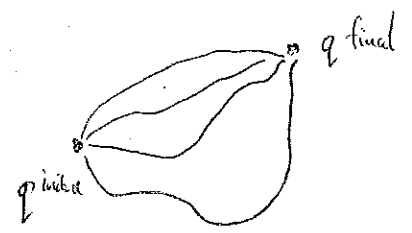
[don't need to explicitly include holonomic constraints]

Note that  $x_i = x_i(q_1, \dots, q_n, t)$

$$dx_i = \sum_j \frac{\partial x_i}{\partial q_j} dq_j + \frac{\partial x_i}{\partial t} dt ; \frac{dx_i}{dt} = \sum_j \frac{\partial x_i}{\partial q_j} \frac{dq_j}{dt} + \frac{\partial x_i}{\partial t}$$

If system moves from  $\vec{q}^{initial} = \{q_1(t_i), \dots, q_n(t_i)\}$  at  $t_i \leftarrow initial$  to  $\vec{q}^{final} = \{q_1(t_f), \dots, q_n(t_f)\}$  at  $t_f \leftarrow final$ .

$$x_i = x_i(q, \dot{q}, t)$$



find motion from Hamilton's principle of least action; integral of the Lagrangian takes minimum possible value between  $t_i$  and  $t_f$ .

"Experience shows" that the state of system is completely described by coordinates + velocities. Then

$$L = L(q_1, \dots, q_n; \dot{q}_1, \dots, \dot{q}_n; t)$$

Hamilton's

principle: Action integral

$$S = \int_{t_i}^{t_f} L(\vec{q}, \dot{\vec{q}}, t) dt \quad \text{is minimum}$$

Assume end points  $\vec{q}^{initial}$  and  $\vec{q}^{final}$  are fixed:  $\delta q(t_i) = \delta q(t_f) = 0$

Vary:

$$\delta S = \int_{t_i}^{t_f} L(q_1 + \delta q_1, \dots, q_n + \delta q_n; \dot{q}_1 + \delta \dot{q}_1, \dots, \dot{q}_n + \delta \dot{q}_n; t) - \int_{t_i}^{t_f} L(q_1, \dots, q_n; \dot{q}_1, \dots, \dot{q}_n; t) dt$$

Expand L to first order

$$\delta S = \sum_j \int_{t_i}^{t_f} \left( \frac{\partial L}{\partial q_j} \delta q_j + \frac{\partial L}{\partial \dot{q}_j} \delta \dot{q}_j \right) dt$$

Put  $\delta \dot{q} = \frac{d}{dt} \delta q$  and integrate by parts <sup>2nd term</sup>

$$\begin{aligned} \text{ie. } \int_{t_i}^{t_f} \frac{\partial L}{\partial \dot{q}} \delta \dot{q} dt &= \int_{t_i}^{t_f} \frac{\partial L}{\partial \dot{q}} \frac{d \delta q}{dt} dt \\ &= \underbrace{\frac{\partial L}{\partial \dot{q}} \delta q}_{=0} \Big|_{t_i}^{t_f} - \int_{t_i}^{t_f} \frac{d}{dt} \left( \frac{\partial L}{\partial \dot{q}} \right) \delta q dt \end{aligned}$$

$$\delta S = \sum_j \int_{t_i}^{t_f} \left( \frac{\partial L}{\partial q_j} - \frac{d}{dt} \left( \frac{\partial L}{\partial \dot{q}_j} \right) \right) \delta q_j dt = 0 \quad (\text{minimum})$$

but this must hold for all  $\delta q_j$

$$\Rightarrow \frac{\partial L}{\partial q_i} - \frac{d}{dt} \left( \frac{\partial L}{\partial \dot{q}_i} \right) = 0$$

Lagrange's equations

for Cartesian coords these are same as Newton's eqns:

put  $L = T - V$ ,  $F_j = -\frac{\partial V}{\partial q_j}$ ,  $T = \sum_j \frac{1}{2} m \dot{q}_j^2$

For velocity independent potentials we get

$$m \ddot{q}_j = -\frac{\partial V}{\partial q_j}$$

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Define generalized momenta

generalized forces

$$p_j = \frac{\partial L}{\partial \dot{q}_j}$$

$$F_j = \frac{\partial L}{\partial q_j}$$

Note that  $p_j$  and  $F_j$  are not ~~in general~~ always components of momentum vector  $\vec{p}$  and force vector  $\vec{F}$

Example: simple pendulum

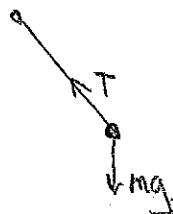
$$L = T - V = \frac{1}{2} m l^2 \dot{\theta}^2 - mgl(1 - \cos\theta)$$

$$\frac{\partial L}{\partial \dot{\theta}} = m l^2 \dot{\theta}$$

$$\frac{\partial L}{\partial \theta} = -mgl \sin\theta$$

Lagrange eqn:

$$-mgl \sin\theta = m l^2 \ddot{\theta}$$



Have not needed to find expressions for polar components of acceleration because we would if we used Newton's laws ~~again~~. Also, have not needed to introduce tension in the rod.

Consider cylindrical polar coords  $\rho, \phi, z$  :  $T = \frac{1}{2} m (\dot{\rho}^2 + \rho^2 \dot{\phi}^2 + \dot{z}^2)$

(depends on time derivatives AND the coords themselves)

$$\frac{\partial T}{\partial \dot{\rho}} = m \dot{\rho}$$

$$\frac{\partial T}{\partial \dot{\phi}} = m \rho^2 \dot{\phi}$$

$$\frac{\partial T}{\partial \dot{z}} = m \dot{z}$$

$$\frac{\partial T}{\partial \rho} = m \rho \dot{\phi}^2$$

$$\frac{\partial T}{\partial \phi} = 0$$

$$\frac{\partial T}{\partial z} = 0$$

Lagrange:  $\frac{d}{dt}(m \dot{\rho}) = m \rho \dot{\phi}^2 - \frac{\partial V}{\partial \rho}$

$$\frac{d}{dt}(m \rho^2 \dot{\phi}) = - \frac{\partial V}{\partial \phi}$$

$$\frac{d}{dt}(m \dot{z}) = - \frac{\partial V}{\partial z}$$

Generalized momenta  $m\dot{\rho}$  and  $m\dot{z}$  are components

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of momentum vector  $\vec{p}$  in the  $\rho$  and  $z$  directions.

But generalized momentum  $m\rho\dot{\phi}$  is the angular momentum  $J_z$  about  $z$ -axis. (not linear)

Beware of notation  $P_\phi$  = does it mean  $\phi$ -component of  $\vec{p}$  or the generalized momentum  $J_z$ ?

$$\frac{dP_\phi}{dt} = F_\phi$$

Only for cartesian coords

do the Lagrange equation coincide with Newton's equations  ~~$\vec{p} = m\vec{v}$~~  i.e. only then do we obtain

$$P_j = m\dot{q}_j$$

Lagrange eqns give  $\dot{p}_j = F_j$

Generalized coords do not usually divide into convenient groups of three that can be associated together to form vectors (Goldstein p 13) i.e. rotate like vectors

More general formulation to include arbitrary forces

i.e. not necessarily of form  $\vec{F} = -\nabla V$

Definition: A conservative force is one that depends only on position (not the path). Typically, we forces depend on velocity.

This time we vary  $I = \int_{t_i}^{t_f} T dt$  where  $T = \frac{1}{2} m \dot{\vec{r}}^2$

$$\frac{1}{2} m \left( \left( \frac{dx}{dt} \right)^2 + \left( \frac{dy}{dt} \right)^2 + \left( \frac{dz}{dt} \right)^2 \right)$$

vary  $x$ -coordinate:  $\delta x(t)$  subject to  $\delta x(t_i) = \delta x(t_f) = 0$

$$\Rightarrow \delta T = m \dot{x} \delta \dot{x}$$

Integrate by parts

$$\delta I = - \int_{t_i}^{t_f} m \dot{x} \delta x dt \quad (\text{boundary terms vanish})$$

but  $-F_x \delta x = -\delta W$   $\delta W$  = work done by force  $\vec{F}$  in displacement  $\delta x$

Vary all 3 coords

$$\delta I = - \int_{t_i}^{t_f} \delta W dt$$

$$\delta W = \vec{F} \cdot \delta \vec{r}$$

Just like Hamilton's principle, and in contrast to Newton's eqns, there is no explicit reference to any particular set of coordinates. Can always write:

$$\delta W = F_1 \delta q_1 + F_2 \delta q_2 + F_3 \delta q_3 \quad F_i = \text{generalised forces}$$

However can also vary T using ~~standard~~ previous method

$$\delta I = \sum_j \int_{t_i}^{t_f} \left[ \frac{\partial T}{\partial q_j} - \frac{d}{dt} \left( \frac{\partial T}{\partial \dot{q}_j} \right) \right] \delta q_j dt$$

Note: here we vary T wrt q and \dot{q} as opposed to Cartesian coords and velocities. (see note below)

equation:

$$\frac{d}{dt} \left( \frac{\partial T}{\partial \dot{q}_j} \right) = \frac{\partial T}{\partial q_j} + F_j$$

If force is conservative, then  $\delta W = -\delta V(q_1, \dots, q_n)$  ( $\delta W = \delta T$  but  $T+V = \text{const}$  so  $\delta T = -\delta V$ )

$$\text{so } F_j = - \frac{\partial V}{\partial q_j}$$

$$\text{(using } \delta W = F_1 \delta q_1 + F_2 \delta q_2 + F_3 \delta q_3)$$

$$\text{then define } L = T - V \Rightarrow \frac{d}{dt} \left( \frac{\partial L}{\partial \dot{q}_j} \right) = \frac{\partial L}{\partial q_j}$$

Note  $T = \frac{1}{2} m (\dot{x}^2 + \dot{y}^2 + \dot{z}^2)$  Cartesian coords.

$$\text{now } \vec{r} = \vec{r}(q_1, \dots, q_n, t) \Rightarrow d\vec{r}_i = \sum_j \frac{\partial \vec{r}_i}{\partial q_j} dq_j + \frac{\partial \vec{r}_i}{\partial t} dt \Rightarrow \frac{d\vec{r}_i}{dt} = \sum_j \frac{\partial \vec{r}_i}{\partial q_j} \frac{dq_j}{dt} + \frac{\partial \vec{r}_i}{\partial t}$$

so  $\vec{r} = \vec{r}(q_1, \dots, q_n; \dot{q}_1, \dots, \dot{q}_n; t)$ , so we can have  $\frac{\partial L}{\partial q_j} \neq 0$  and  $\frac{\partial L}{\partial \dot{q}_j} \neq 0$ .

# Example

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## 2D Pendulum



Assume that we also apply torque  $\mathcal{T}(\phi)$  for some time

$$L = \frac{1}{2} m l^2 (\dot{\theta}^2 + \dot{\phi}^2 \sin^2 \theta) - mgl(1 - \cos \theta)$$

work done by torque  $\delta W = \mathcal{T} \delta \phi$

Definition: No. of coords that can vary independently is No. of degrees of freedom. Here: 2 deg. of freedom.  $(\theta, \phi)$

Lagrange eqns

$$\frac{d}{dt} \left( \frac{\partial L}{\partial \dot{\theta}} \right) = \frac{\partial L}{\partial \theta}, \quad \frac{d}{dt} \left( \frac{\partial L}{\partial \dot{\phi}} \right) = \frac{\partial L}{\partial \phi} + \mathcal{T}$$

$$\Rightarrow m l^2 \ddot{\theta} = m l^2 \dot{\phi}^2 \sin \theta \cos \theta - mgl \sin \theta$$

$$\frac{d}{dt} (m l^2 \dot{\phi} \sin^2 \theta) = \mathcal{T}$$

Now suppose  $\mathcal{T}$  is adjusted to constrain the system to rotate with const. ang. speed  $\omega$  about vertical

$$\text{constraint: } \dot{\phi} = \omega$$

Now only 1 degree of freedom:  $\theta$

sub. constraint into  $L$ :

$$L = \frac{1}{2} m l^2 (\dot{\theta}^2 + \omega^2 \sin^2 \theta) - mgl(1 - \cos \theta)$$

$$= T' - V'$$

relative:

$$T' = \frac{1}{2} m l^2 \dot{\theta}^2$$

quadratic  
in  $\dot{\theta}$

$$V' = \underbrace{mgl(1 - \cos \theta) - \frac{1}{2} m l^2 \omega^2 \sin^2 \theta}_{\text{independent of } \dot{\theta}}$$

centrifugal term

Same as going into rotating frame

In a forced system, constraining forces can do work

so  $T+V \neq \text{const.}$

But there still is a conservation law:

multiply  $ml^2\ddot{\theta} = ml^2\dot{\phi}^2 \sin\theta \cos\theta - mgl \sin\theta$  by  $\dot{\theta}$

integrate over time  $\Rightarrow T'+V' = E' = \text{const.}$

this is not total energy  $T+V$  because centrifugal term appears with opposite sign.

$T'+V' = T+V - ml^2\omega^2 \sin^2\theta.$

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Details :

$\int ml^2\ddot{\theta} \times \dot{\theta} dt = \frac{1}{2} ml^2 \int \frac{d}{dt}(\dot{\theta}^2) dt = \frac{1}{2} ml^2 \dot{\theta}^2 + \text{const}$   
+ const of integration

$\int \left\{ ml^2\omega^2 \underbrace{\sin\theta \cos\theta}_{\frac{1}{2} \frac{d}{d\theta} \sin^2\theta} - mgl \sin\theta \right\} \frac{d\theta}{dt} dt$

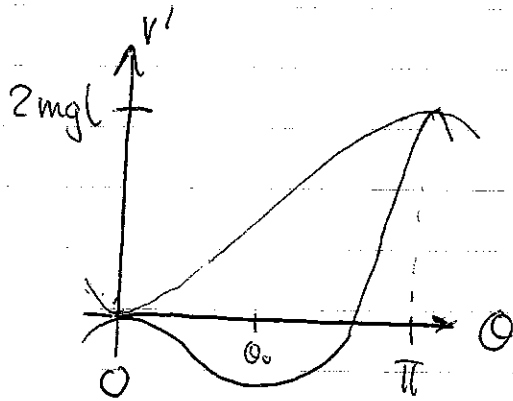
$= \frac{1}{2} ml^2\omega^2 \sin^2\theta + mgl \cos\theta + \text{const}$

So  $\underbrace{\frac{1}{2} ml^2 \dot{\theta}^2}_{T'} - \underbrace{\left( \frac{1}{2} ml^2\omega^2 \sin^2\theta - mgl \cos\theta \right)}_{+V'} = \text{const}$

cf  $T+V = \frac{1}{2} ml^2(\dot{\theta}^2 + \dot{\phi}^2 \sin^2\theta) + mgl(1 - \cos\theta)$

# Effective potential $V'$

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upper  
curve

$$\omega^2 < \frac{g}{L}$$

still  
 $V'$  has min at  $\theta = 0$

motion qualitatively same as ordinary pendulum  
but period is longer

lower  
curve

$$\omega^2 > \frac{g}{L}$$

$V'$  has minimum at  $\theta = \theta_0 = \arccos\left(\frac{g}{L\omega^2}\right)$

At  $\theta_0$  gravity and centrifugal force are  
in equilibrium

3 types of motion :

$E' < 0$  pendulum oscillates about  $\theta_0$

$0 \leq E' < 2mgl$  swings backwards and forwards  
as before though  $\theta = 0$  is no longer  
position of maximum velocity.

$E' > 2mgl$  "windmill" i.e. continuous  
circular motion



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We call the constraint  $\dot{\phi} = \omega$  a holonomic constraint

Definition : A system is holonomic if it is possible to solve the constraint eqns, and <sup>so</sup> eliminate some of the co-ordinates, leaving a set of general coordinates equal to the number of degrees of freedom.

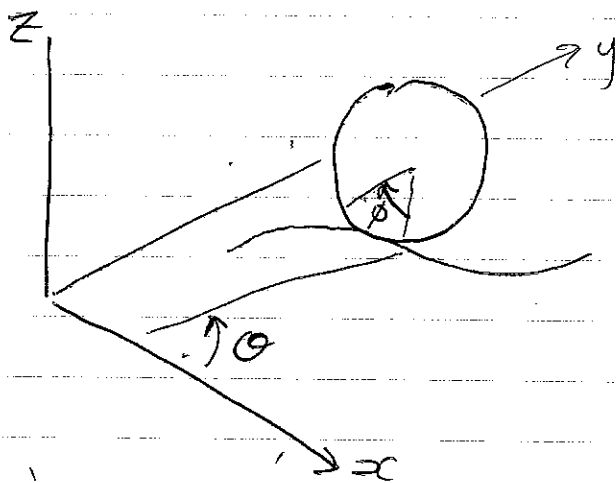
Here we introduced a constraint,  $\dot{\phi} = \omega$ , and the problem was reduced to 1 degree of freedom and one generalized coordinate,  $\theta$ .

Counter ~~ex~~ example: (non-holonomic)

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Disk rolling on horizontal plane ( $z=0$ )

Coords:  $(x, y, \phi, \omega)$   
 $\phi$ : angle between disk and, e.g.  $x$ -axis  
 $\omega$ : angle of rotation about axis of disk



Rolling constraint:  $v = a \dot{\phi}$        $a = \text{radius}$

$$v_x = \dot{x} = v \sin \phi$$

$$v_y = \dot{y} = -v \cos \phi$$

Combine to give two differential equations of constraint:

$$dx - a \sin \phi d\phi = 0$$

$$dy + a \cos \phi d\phi = 0$$

Cannot integrate either without first solving the problem.

(cannot find integrating factor  $f(x, y, \phi, \omega)$  that turns either eqns into perfect differentials). Thus: cannot solve constraint equation and eliminate some coords.

Physical reason: no direct functional relation between  $\phi$  and  $(x, y, \omega)$

because can ~~also~~ always roll disk in circle of arbitrary radius.

$x, y, \omega$  return to original values but  $\phi$  depends on radius. Thus need to know precise path - state of system not just function of "position"  $(x, y, \omega)$  - still need  $\phi$ .

More general method for constraints (holonomic and nonholonomic)

Method of Lagrange multipliers:  $\lambda$

$$L = \frac{1}{2} m l^2 (\dot{\theta}^2 + \dot{\phi}^2 \sin^2 \theta) - mgl(1 - \cos \theta) - \lambda(\dot{\phi} - \omega)$$

EL for  $\theta$  unchanged

EL for  $\phi$

$$\frac{d}{dt} (m l^2 \dot{\phi} \sin^2 \theta - \lambda) = 0$$

EL for  $\lambda$

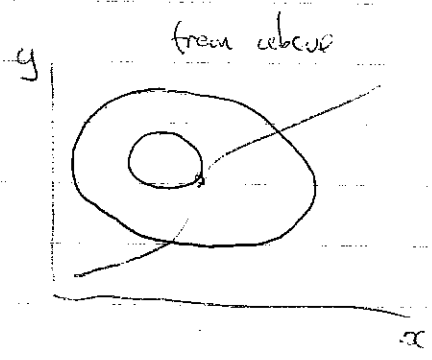
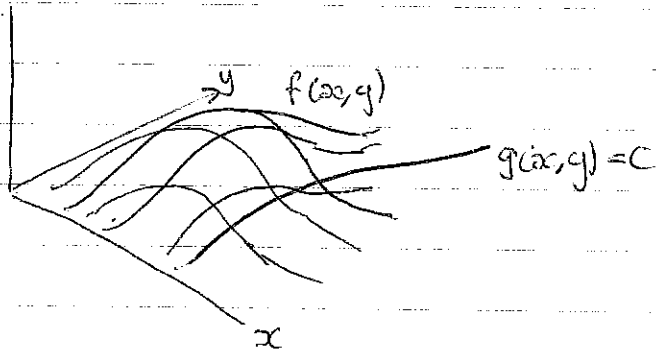
$$0 = \dot{\phi} - \omega$$

(reproduces constraint.)

$$\lambda = \tau \quad \text{torque}$$

This is an example of finding a minimum or maximum of a function subject to constraints. Example, minimize  $f(x,y)$  subject to  $g(x,y) = c$

Consider  $A(x,y,\lambda) = f(x,y) + \lambda(g(x,y) - c)$



## General idea

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Walk along  $g=c$ .

In general  $g=c$  intersects contours of  $f$ , i.e.  $f$  varies along  $g=c$ .

However when contour  $g=c$  meets a contour of  $f$  tangentially, then we don't ~~see~~ increase or decrease of  $f$  either.

Contours of  $f$  and  $g$  touch when their tangent vectors are parallel, and so their gradients are also parallel.

Need:  $\vec{\nabla}_{x,y} f = -\lambda \vec{\nabla}_{x,y} g$  (magnitudes not generally equal.)

$$g = c$$

so solve  $\vec{\nabla}_{x,y,d} A(x,y,d) = 0$